

IN THE SPECIFICATION:

Paragraph beginning at line 3 page 1 has been amended as follows:

The present invention relates to a vane rotary type gas compressor for use in an automotive air conditioning system or the like and, in particular, to a vane rotary type gas compressor improved in terms of vane projectability at the time start up of operation ~~start~~ of the compressor.

Paragraph beginning at line 13 of page 3 has been amended as follows:

Thus, in the compressor main body 1 and the cylinder 3, there is provided a supplying system for supplying lubricant. The lubricant supplying system in the compressor main body 1 and the cylinder 3 will be described. Lubricant is stored in an oil sump 7 formed in the lower portion of the exhaust chamber 6. The lubricant stored in the oil sump 7 is supplied to the various portions mentioned above. More specifically, lubricant is supplied to a plain bearing 9a in the rear side block 9 and a plain bearing 8a in the front side block 8. Further, lubricant is supplied to flat, arcuate grooves 11 formed in the rear side block 9 and the front side block 8 so as to be opposed to the rotor 4 and adapted to communicate with one of the plurality of vane grooves 16 when

the rotating angle of the rotor 4 is within a fixed angle range. Further, lubricant is supplied to a high pressure supplying hole 10 formed in the rear side block 9 so as to be opposed to the rotor 4 and adapted to communicate with one of the plurality of vane grooves 16 when the rotating angle of the rotor 4 is within a fixed angle range. Further, lubricant is supplied to the compression chambers 5a and other sliding portions. At this time, the flat groove 11 and the high pressure supplying hole 10 are spaced apart from each other to a degree such that they do not communicate with each other through the vane grooves 16.

Paragraph beginning at line 7 of page 3 has been amended as follows:

As described above, the gas compressor sucks in and compresses refrigerant gas, so that it necessary to effect lubrication and sealing on plain bearings and other sliding portions, etc. in the compressor main body 1 and on the sliding portions such as the rotor 4 and the vanes 17 and the compression chambers 5a in the cylinder 3, and lubricant is used for that purpose.

Paragraph beginning at line 3 of page 11 has been amended as follows:

To achieve the above object, in accordance with the present invention, there is provided a gas compressor for sucking in, compressing, and discharging refrigerant gas, characterized by ~~including~~ an elliptical cylinder, a rotor rotatably arranged in the cylinder, vane grooves radially formed in the rotor, vanes provided in the vane grooves and capable of projecting and retracting radially with respect to the rotor, a flat groove adapted to communicate with vane groove bottom portions during a refrigerant gas sucking/compressing process, a high pressure supplying hole adapted to communicate with the vane groove bottom portions upon interception of the communication between the vane groove bottom portions and the flat groove in the refrigerant gas compressing process, and a communication passage adapted to establish communication between the flat groove and the high pressure supplying hole at the start of the gas compressor.

Paragraph beginning at line 6 of page 12 has been amended as follows:

Further, in accordance with the present invention, the gas compressor is characterized by ~~further including~~ a discharge chamber for temporarily storing refrigerant gas discharged from the cylinder, an oil sump formed in a lower

portion of the exhaust chamber, a first supplying passage establishing communication between the oil sump and the high pressure supplying hole, and a second supplying passage branching off from the first supplying passage and communicating with the flat groove, the communication passage being formed by the first supplying passage and the second supplying passage.

Paragraph beginning at line 5 of page 17 has been amended as follows:

Fig. 1 is a longitudinal sectional view showing a first embodiment of the gas compressor of this invention. Figs. 2a 2A and 2B are schematic diagrams showing a communication passage and a lubricant supplying passage according to this embodiment.

Paragraph beginning at line 1 of page 18 has been amended as follows:

In this embodiment, there is provided a second supply passage 14 further branching off from the ~~branches off point 12b~~ branch-off point 12a for the first supplying passage 12 and the third supplying passage 13, formed inside the rear side block 9, and establishing communication between the high pressure supplying hole 10 and the flat groove 11.

Paragraph beginning at line 6 of page 23 has been amended as follows:

As described above, in accordance with this embodiment, communication is established between the high pressure supplying hole 10 and the flat groove 11 by the communication passage 21, whereby the projectability of the vanes 17 at the start of the compressor is dramatically improved, and the vanes 17 ~~divides~~ divide the cylinder chamber 5 immediately after the start of the compressor to define the compression chambers 5a, sucking and compressing refrigerant gas. Thus, no matter how adverse the conditions are, the requisite starting performance is ensured, and chattering, etc., at the time of starting is prevented as well.

Paragraph beginning at line 15 of page 27 has been amended as follows:

In addition to the above construction, this second embodiment adopts a construction in which there is provided a second pressure control valve 20 at a position in the first supplying passage 12 on the downstream side of the oil sump 7 and on the upstream side of the ~~branches-off~~ branch-off points 12a and 12b for the second supplying passage 14 and the third supplying passage 13.

Paragraph beginning at line 3 of page 28 has been amended as follows:

When the gas compressor is at rest, there is no high pressure refrigerant gas to be discharged into the exhaust chamber 6, so that the pressure in the exhaust chamber 6 is lower than that during normal operation of the gas compressor. At this time, the difference between the pressure in the exhaust chamber 6 and the pressure at the ~~branches-off~~ branch-off point 12a for the second supplying passage 14 is not larger than a predetermined value, and the second pressure control valve 20 keeps the first supplying passage 12 in the closed state, interrupting the first supplying passage 12.

Paragraph beginning at line 15 of page 31 has been amended as follows:

Next, in this second embodiment, it is possible to further provide in the third supplying passage 13 a third pressure control valve 24 which is of the same construction and operation as the second pressure control valve 20 and which is operative or adapted to be brought into the closed state when the difference between the pressure of the exhaust chamber 6 and the pressure in the third supplying passage 13 is not larger than a predetermined value.

Paragraph beginning at line 3 of page 33 has been amended as follows:

In this third embodiment, the second supplying passage 14 branches off from the first supplying passage 12 at a point on the downstream side of the oil sump 7 and on the downstream side of the ~~branches-off~~ branch-off point for the first supplying passage 12 and the third supplying passage 13. Further, the second pressure control valve 20 of this embodiment is provided in the first supplying passage 12 at a position between the ~~branches-off~~ branch-off point 12a for the second supplying passage 14 and the ~~branches-off~~ branch-off point 12b for the third supplying passage 13.